Topology optimisation of heat sinks for instantaneous chip cooling: using a transient pseudo-3D thermofluid model

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Motivation

- \rightarrow Increasing power density of electronics
 - \rightarrow Smaller and more powerful
 - \rightarrow Serious thermal management issues
- \rightarrow Electronics is inherently transient
 - \rightarrow Varying thermal power output
 - ightarrow Needs fast response for sudden peak in load

160 0.09 0.085 155 0.08 150 0.075 145 0.07 140 0.065 135 0.06 130 0.055 Temperature [C] اءًا 125 0.05 [W/ 120 0.045 Power 0.04 115 0.035 110 0.03 105 0.025 100 0.02 95 Temperature 0.015 90 0.01 85 0.005 Powe 80 L 0 SDU 🎓 100 20 40 60 80 Time (s)



[https://www.pikist.com/free-photo-suljg/da]



Instantaneous cooling

[Zeng, ..., Alexandersen – doi:10.1016/j.ijheatmasstransfer.2020.119681]

- \rightarrow CPU chip: maximum temperature limit
 - \rightarrow Above limit, speed is throttled down (worse performance)
 - \rightarrow Critical case: turn on cooling system when limit is reached
- \rightarrow Goal: lower temperature fast!
 - → Not *long* time frame: steady state performance
 - → *Short* time frame: instantaneous performance
- \rightarrow Requires time-dependent simulation model
 - → Computationally expensive!

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→ Simplified model to make each time step cheaper?





[https://www.pikist.com/free-photo-suljg/da]



Transient flow-based problems

 \rightarrow Recent review of topology optimisation of flow-based problems:

 \rightarrow showed that only 7% of 186 papers treated transient problems!



(a) All papers

(b) Fluid flow only

Figure 8. Distribution of papers for fluid model type: SS = steady-state laminar flow; TR = transient laminar flow; TU = turbulent flow; NN = Non-Newtonian fluid.

[Alexandersen & Andreasen – doi:10.3390/fluids5010029]





#sdudk



Pseudo-3D: 2 layer model

[Zeng, ..., Alexandersen – doi:10.1016/j.ijheatmasstransfer.2020.119681]

→ Pseudo-3D model by Haertel et al. [doi:10.1016/j.ijheatmasstransfer.2018.01.078]

- \rightarrow Extruded heat sink geometry with base plate (heat source)
- \rightarrow Cut-plane through channel and heat sink
- \rightarrow Channel layer is then coupled to base plate layer
- → Heuristic coupling heat transfer coefficient
- → Zeng et al. [doi:10.1016/j.ijheatmasstransfer.2018.01.039]
 - \rightarrow Calculate coupling coefficient from reference geometry
- → Alternative: Yan et al. [doi:10.1016/j.ijheatmasstransfer.2019.118462]
 - \rightarrow Analytical derivation based on assumptions



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Instantaneous performance

[Zeng, ..., Alexandersen – doi:10.1016/j.ijheatmasstransfer.2020.119681]



Steady-state performance

[Zeng, ..., Alexandersen – doi:10.1016/j.ijheatmasstransfer.2020.119681]





But steady-state design outperforms transient design for long term cooling!



Water-cooled design

[Zeng, ..., Alexandersen – doi:10.1016/j.ijheatmasstransfer.2020.119681]





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Water-cooled design: practical conditions

[Zeng, ..., Alexandersen – doi:10.1016/j.ijheatmasstransfer.2020.119681]



→ Hits 90°C, cooling system turned on → Reduced to 55°C, cooling system turned off



 \rightarrow Time from 90 to 55 much faster!

ightarrow Time from 55 back to 90 more or less the same

→ Pumping energy: 67% saved!

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Any questions?



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